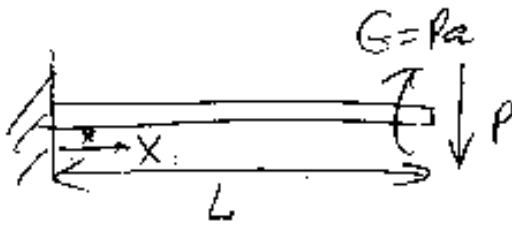


4ME06 Stress Analysis

①



Total energy:
$$W = \underbrace{\frac{1}{2}EI \int_0^L (y'')^2 dx}_{\text{REMEMBER!}} - Py(L) - Gy'(L)$$

Analytical treatment to obtain exact soln:

$$\begin{aligned} \delta W &= 0 \\ \delta W &= EI \int_0^L y'' \delta y'' dx - P \delta y(L) - G \delta y'(L) = \\ &= -EI \int_0^L y''' \delta y' dx + [EI y'' \delta y']_0^L - P \delta y(L) - G \delta y'(L) = \\ &= EI \int_0^L y'''' \delta y dx - [EI y'''' \delta y]_0^L + [EI y''' \delta y']_0^L - P \delta y(L) - G \delta y'(L) \end{aligned}$$

B.c.: for $x=0$ $\delta y = \delta y' = 0$, since $y = y' = 0$ @ $x=0$

Hence for $\delta W = 0$ require

- (i) $EI y'''' = 0$, $0 < x < L$ - this is EULER.
 - (ii) $P = -EI y''''(L)$, since $\delta y(L)$ is arbitrary
 - (iii) $G = EI y'''(L)$, since $\delta y'(L)$ is arbitrary
- } Nat'l b.c.

Hence $y = A + Bx + Cx^2 + Dx^3$.

$x=0, y=y'=0 \Rightarrow A=B=0$

$x=L, y'' = 2C + 6DL = G/EI = Pa/EI$

$x=L, y''' = 6D = -P/EI$

$C = \frac{PL + Pa}{2EI}$
$D = -\frac{P}{6EI}$

EXACT

(a) Rayleigh - Ritz :

$$y = A + Bx + Cx^2 + Dx^3$$

$$\text{B.c.'s } y(0) = y'(0) = 0 \Rightarrow A = B = 0$$

$$y'' = 2C + 6Dx$$

$$W = \frac{1}{2} EI \int_0^L (y'')^2 dx - Py(L) - Gy'(L) =$$

$$= \frac{1}{2} EI \int_0^L (4C^2 + 24CDx + 36D^2x^2) dx - P(CL + DL^3) - G(2CL + 3DL^2)$$

$$W = 2EI(C^2L + 3CDL^2 + 3D^2L^3) - P(CL^2 + DL^3) - G(2CL + 3DL^2)$$

$$\frac{\partial W}{\partial C} = 0 = 2EIL(2C + 3DL) - PL^2 - 2GL$$

$$\frac{\partial W}{\partial D} = 0 = 2EIL(3CL + 6DL^2) - PL^3 - 3GL^2$$

$$\boxed{C = \frac{PL + Pa}{2EI} \quad D = -\frac{P}{6EI}}$$

Exact solution obtained, since sufficient series terms.

Natural b.c.'s:

$$M(L) = EIy''(L) = EI(2C + 6DL) = PL + Pa - PL = Pa$$

OK.

$$(b) \quad y = A \left(1 - \cos \left(\frac{\pi x}{2L} \right) \right)$$

$$y' = \frac{A\pi}{2L} \sin \left(\frac{\pi x}{2L} \right)$$

$$y'' = A \left(\frac{\pi}{2L} \right)^2 \cos \left(\frac{\pi x}{2L} \right)$$

$$y''' = -A \left(\frac{\pi}{2L} \right)^3 \cos \left(\frac{\pi x}{2L} \right)$$

$$W = \frac{1}{2} EI A^2 \left(\frac{\pi}{2L} \right)^4 \int_0^L \cos^2 \left(\frac{\pi x}{2L} \right) dx - PA - G \frac{A\pi}{2L} =$$

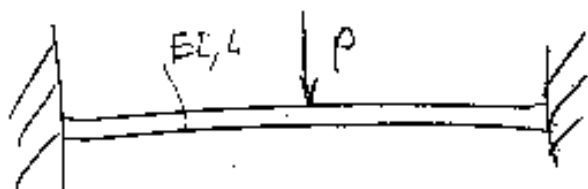
$$= \frac{1}{4} EI A^2 \left(\frac{\pi}{2L} \right)^4 - PA - G \frac{A\pi}{2L}$$

$$\frac{\partial W}{\partial A} = 0 = \frac{2AEI}{4} \left(\frac{\pi}{2L} \right)^4 - P - \frac{G\pi}{2L}; \quad \boxed{A = \frac{32PL^2}{\pi^4 EI} \left(1 + \frac{G\pi}{2L} \right)}$$

NB: $M(L) = EI y''(L) = 0$ - Natural b.c. not satisfied.

Cf. $y(L)$ (a) Polynomial (exact) $\frac{1}{EI} \{ 0.333 PL^3 + 0.5 PaL^2 \}$
 (b) Curve $\frac{1}{EI} \{ 0.323 PL^3 + 0.516 PaL^2 \}$

②



$$\text{Let } y = A \left(1 - \cos \frac{2\pi x}{L}\right)$$

$$y' = \left(\frac{2\pi}{L}\right) A \sin \left(\frac{2\pi x}{L}\right)$$

$$y'' = \left(\frac{2\pi}{L}\right)^2 A \cos \frac{2\pi x}{L} \quad y'''' = -\left(\frac{2\pi}{L}\right)^4 A \cos \frac{2\pi x}{L}$$

$$W = \frac{1}{2} EI \left(\frac{2\pi}{L}\right)^4 A^2 \int_0^L \cos^2 \left(\frac{2\pi x}{L}\right) dx - PA(1 - \cos \pi) =$$

$$= \frac{4\pi^4 A^2}{L^3} EI - 2PA$$

$$\frac{\partial W}{\partial A} = \frac{8\pi^4 A EI}{L^3} - 2P = 0 \quad \Rightarrow \quad A = \frac{PL^3}{4\pi^4 EI}$$

$$y(L/2) = 2A = \frac{PL^3}{2\pi^4 EI}$$

$$M(L/2) = EI y''(L/2) = \frac{PL}{\pi^2}$$

NB: GALEKIN gives the same result, since EULER is:

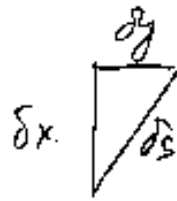
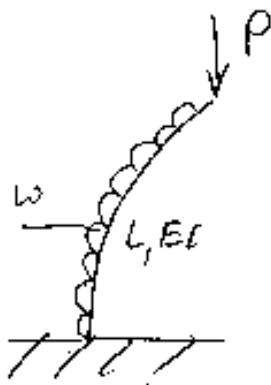
$$EI y'''' - P\delta(x - L/2) = 0$$

$$\text{GAL: } \int_0^L [EI y'''' - P\delta(x - L/2)] (1 - \cos \frac{2\pi x}{L}) dx = 0 =$$

$$= \left(\frac{2\pi}{L}\right)^4 EI \int_0^L \cos \frac{2\pi x}{L} (1 - \cos \frac{2\pi x}{L}) dx - P(1 - \cos \pi)$$

$$\Rightarrow A = \frac{PL^3}{4\pi^4 EI} \text{ again.}$$

(3)



$$\delta s^2 = \delta x^2 + \delta y^2$$

$$\delta s = dx \sqrt{1 + \left(\frac{\delta y}{\delta x}\right)^2}$$

$$\Delta = \int_0^L ds - L = \int_0^L \left[1 + \frac{1}{2} \left(\frac{dy}{dx}\right)^2\right] dx - L = \frac{1}{2} \int_0^L \left(\frac{dy}{dx}\right)^2 dx$$

$$W = \frac{1}{2} EI \int_0^L \left(\frac{dy}{dx}\right)^2 dx - \frac{1}{2} P \int_0^L \left(\frac{dy}{dx}\right)^2 dx - \int_0^L \omega y dx$$

(a) Take $y = A \left(1 - \cos\left(\frac{\pi x}{2L}\right)\right)$

$$y' = A \frac{\pi}{2L} \sin \frac{\pi x}{2L}$$

$$y'' = A \left(\frac{\pi}{2L}\right)^2 \cos \frac{\pi x}{2L}$$

$$W = \frac{1}{2} EI \int_0^L A^2 \left(\frac{\pi}{2L}\right)^2 \cos^2 \frac{\pi x}{2L} dx - \frac{1}{2} P \int_0^L A^2 \left(\frac{\pi}{2L}\right)^2 \sin^2 \frac{\pi x}{2L} dx - \omega \int_0^L A \left(1 - \cos \frac{\pi x}{2L}\right) dx$$

$$W = \frac{EI}{64L^3} A^2 \pi^4 - \frac{A^2 \pi^2 P}{16L} - \frac{2A\omega L}{\pi}$$

$$\frac{\partial W}{\partial A} = \frac{EI \pi^4}{32L^3} A - \frac{\pi^2 P}{8L} A - \frac{2\omega L}{\pi} = 0$$

$$A = \frac{2\omega L}{\pi} \sqrt{\frac{EI \pi^4}{32L^3} - \frac{\pi^2 P}{8L}}$$

$$(b) \quad y = A + Bx + Cx^2 + Dx^3$$

B.C's: $x=0, y=y'=0 \Rightarrow A=B=0$

$$y = Cx^2 + Dx^3$$

$$y' = 2Cx + 3Dx^2$$

$$y'' = 2C + 6Dx$$

$$W = 2EIL \left[C^2 + 3CDL + 3D^2L^2 \right] - \frac{P}{2} \left[\frac{4}{3}CL^3 + 3CDL^4 + \frac{9}{5}D^2L^5 \right]$$

$$- W \left(\frac{CL^3}{3} + \frac{DL^4}{4} \right)$$

Eq. A: $\frac{\partial W}{\partial C} = 0 = 4EILC + 6EIDL^2 - \frac{4}{3}PL^3 - \frac{3}{2}PDL^4 - \frac{W}{3}L^3$

Eq. B: $\frac{\partial W}{\partial D} = 0 = 2EIL(3CL + 6DL^2) - \frac{P}{2}(3CL^4 + \frac{18}{5}DL^5) - \frac{WL^4}{4}$

From Eq. A $\rightarrow C = \frac{\frac{WL^3}{3} + D \left[\frac{3}{2}PL^4 - 6EIL^2 \right]}{4EIL - \frac{4}{3}PL^3}$

From Eq. B $\rightarrow 2EIL \left[\frac{3L \left(\frac{WL^3}{3} + D \left(\frac{3}{2}PL^4 - 6EIL^2 \right) \right)}{4EIL - \frac{4}{3}PL^3} + 6DL^2 \right] -$

$$- \frac{P}{2} \left[\frac{3L^4 \left(\frac{WL^3}{3} + D \left(\frac{3}{2}PL^4 - 6EIL^2 \right) \right)}{4EIL - \frac{4}{3}PL^3} \right] + \frac{18}{5}DL^5 - \frac{WL^4}{4} = 0$$

Gives C, D

(c) Use natural b.c.:

$$M(L) = EI y''(L) = 2C + 6DL = 0; D = -\frac{C}{3L}$$

$$W = 2EIC^2L \left[1 + 3L \left(-\frac{1}{3L}\right) + \frac{3L^2}{9L^2} \right] - \frac{PC^2}{2} \left[\frac{4}{3}L^3 - \frac{3L^4}{3L} + \frac{2L^5}{5(3L)^2} \right] - \\ - WCL^3 \left(\frac{1}{3} - \frac{1}{12} \right) = \frac{2EIC^2L}{3} - \frac{4}{15} PC^2L^3 - \frac{WCL^3}{4}$$

$$\frac{\partial W}{\partial C} = 0 = \frac{4EICL}{3} - \frac{8}{15} PCL^3 - \frac{WL^3}{4};$$

$$C = \frac{WL^3}{4} / \left[\frac{4EICL}{3} - \frac{8}{15} PCL^3 \right]$$

(4)

$$W = \frac{1}{2} D \iint \left(\frac{\partial w}{\partial x^2} + \frac{\partial w}{\partial y^2} \right)^2 dx dy - \omega(\xi, \eta) P$$

$$w = \sum \sum a_{mn} \sin \frac{m\pi x}{a} \sin \frac{n\pi y}{b}$$

$$w_{xx} = \sum \sum a_{mn} \left(\frac{m\pi}{a} \right)^2 \sin \frac{m\pi x}{a} \sin \frac{n\pi y}{b}$$

$$w_{yy} = \sum \sum a_{mn} \left(\frac{n\pi}{b} \right)^2 \sin \frac{m\pi x}{a} \sin \frac{n\pi y}{b}$$

Rayleigh - Ritz:

$$\frac{\partial W}{\partial a_{ij}} = 0 = D \iint (w_{xx} + w_{yy}) \left(\frac{\partial w_{xx}}{\partial a_{ij}} + \frac{\partial w_{yy}}{\partial a_{ij}} \right) dx dy - P \frac{\partial \omega(\xi, \eta)}{\partial a_{ij}}$$

$$0 = D \iint \left[\sum \sum a_{mn} \left(\frac{m\pi}{a} \right)^2 \sin \frac{m\pi x}{a} \sin \frac{n\pi y}{b} + \sum \sum a_{mn} \left(\frac{n\pi}{b} \right)^2 \sin \frac{m\pi x}{a} \sin \frac{n\pi y}{b} \right] \times$$

$$\times \left[\left(\frac{i\pi}{a} \right)^2 \sin \frac{i\pi x}{a} \sin \frac{j\pi y}{b} + \left(\frac{j\pi}{b} \right)^2 \sin \frac{i\pi x}{a} \sin \frac{j\pi y}{b} \right] dx dy -$$

$$- P \sin \frac{i\pi \xi}{a} \sin \frac{j\pi \eta}{b} = 0$$

Due to orthogonality:

$$0 = D \iint a_{ij} \sin^2 \frac{i\pi x}{a} \sin^2 \frac{j\pi y}{b} \left[\left(\frac{i\pi}{a} \right)^2 + \left(\frac{j\pi}{b} \right)^2 \right] dx dy - P \sin \frac{i\pi \xi}{a} \sin \frac{j\pi \eta}{b}$$

If $P=1$, then we find

$$a_{mn} = \frac{4 \sin \frac{m\pi \xi}{a} \sin \frac{n\pi \eta}{b}}{D a b \pi^4 \left[\left(\frac{m}{a} \right)^2 + \left(\frac{n}{b} \right)^2 \right]^2}$$

(5)

$$W = \int_0^r \frac{D}{2} 2\pi \left[\omega_{rr}^2 + \frac{1}{r^2} \omega_r^2 + \frac{2D}{r} \omega_r \omega_{rr} \right] r dr - \int_0^a 2\pi r \rho_0 \omega dr$$

Given that $D = D_0 \left(1 - \frac{r}{2a}\right)$

and using $\omega = \omega_0 \left(1 - \left(\frac{r}{a}\right)^2\right)$

$$\omega' = \omega_r = -2\omega_0 r / a^2$$

$$\omega_{rr} = -2\omega_0 / a^2$$

$$W = \pi \int_0^a D_0 \left(1 - \frac{r}{2a}\right) \left(\frac{4\omega_0^2}{a^4} + \frac{4\omega_0^2}{a^4} + \frac{8D\omega_0^4}{a^4} \right) r dr - 2\pi \rho_0 \omega_0 \int_0^a \left(1 - \left(\frac{r}{a}\right)^2\right) r dr$$

$$W = \frac{8}{3} \frac{\pi D_0 \omega_0^2}{a^2} (1+\nu) - \frac{1}{2} \pi \rho_0 \omega_0 a^2$$

$$\frac{\partial W}{\partial \omega_0} = \frac{16}{3} \frac{\pi D_0 \omega_0}{a^2} (1+\nu) - \frac{1}{2} \pi \rho_0 a^2 = 0$$

$$\Rightarrow \boxed{\omega_0 = \frac{3}{32} \frac{\rho_0 a^4}{D_0}}$$

$$\textcircled{6} \text{ Equilibrium: } b_{xy,x} + \tau_{xy,y} + p_x = 0 \quad (x)$$

$$b_{yx,y} + \tau_{yx,x} + p_y = 0 \quad (y)$$

$$\text{Compatibility: } \epsilon_{xx,yy} + \epsilon_{yy,xx} - 2\epsilon_{xy,xy} = 0$$

$$\text{Hooke: } \epsilon_{xx} = \frac{1}{E} [b_{xx} - \nu b_{yy}]$$

$$\epsilon_{yy} = \frac{1}{E} [b_{yy} - \nu b_{xx}]$$

$$\epsilon_{xy} = \frac{1}{2G} \tau_{xy} = \frac{1+\nu}{E} \tau_{xy}$$

From equil. by diff:

$$b_{xy,xx} + \tau_{xy,xy} + p_{x,x} = 0$$

$$b_{yx,yy} + \tau_{yx,xy} + p_{y,y} = 0$$

$$\text{Into compat: } \frac{1}{E} [b_{xx,yy} + b_{yy,xx} - \nu b_{xy,xy} - \nu b_{yx,xy}] +$$

$$+ \frac{1+\nu}{E} [p_{x,x} + p_{y,y} + b_{xx,xx} + b_{yy,yy}] = 0$$

$$\frac{1}{1+\nu} [b_{xx,yy} + b_{yy,xx} - \nu(b_{xx,xx} + b_{yy,yy}) + (1+\nu)(b_{xx,xx} + b_{yy,yy})] = -p_{x,x} - p_{y,y}$$

$$\text{Hence } \nabla^2(b_{xx} + b_{yy}) = -(1+\nu)(p_{x,x} + p_{y,y}).$$

(7) Postulate displacement field:

(i) $u = \frac{\delta}{H} v \frac{r}{R} f(z/H), v=0, w = -\frac{\delta}{H} z$

let $z/H = s$; $f(s)$ - bending function; $e = \frac{H}{R}$

$u = \frac{e v \delta r}{H^2} f(s); v=0; w = -\frac{\delta}{H} z; \Rightarrow$ need $f(\pm 1) = 0$

(ii) $\epsilon_{rr} = \frac{\partial u}{\partial r} = \frac{e v \delta}{H^2} f(s); \epsilon_{\theta\theta} = \frac{u}{r} = \epsilon_{rr}; \epsilon_{zz} = -\frac{\delta}{H}; \epsilon_{rz} = \frac{\delta v e r f'(s)}{2H^2}$

Note also ϵ_{rz} is non-zero \Rightarrow shear at platens.

(iii) $b_{rr} = -\frac{\delta v (H - e f(s)) E}{H^2 (1+\nu)(1-2\nu)}; b_{zz} = -\frac{\delta (H - H\nu - 2e v^2 f(s)) E}{H^2 (1+\nu)(1-2\nu)}$

Traction at top/bottom surface: note $f(\pm 1) = 0!$

$b_{rz}|_{z=\pm H} = -\frac{\delta (1-\nu) E R^2}{H(1+\nu)(1-2\nu)}; t_{ext} = \frac{2\delta^2 (1-\nu) E H}{e^2 (1+\nu)(1-2\nu)}$

(iv) Use strain energy: solve based on KA disp field!

Need matrices to perform calcs! see cyl 2.4b